

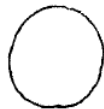
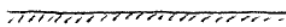
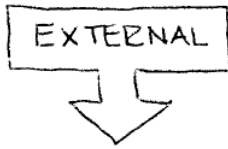
**NOTES: Internal convection**

TWO TYPES OF

*Flow* :

• EXTERNAL

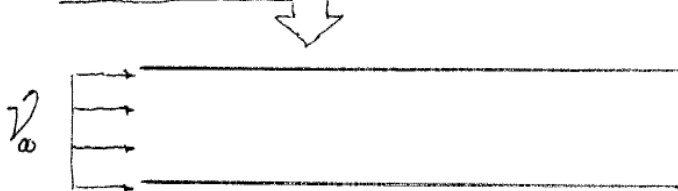
• INTERNAL



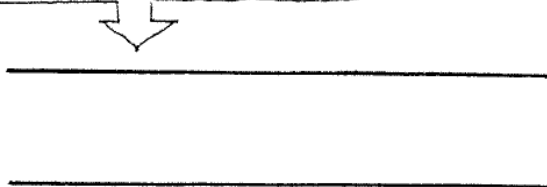
**INTERNAL FLOW**

How does internal flow differ from external flow in terms of boundary layers?

VELOCITY (MOMENTUM) BOUND. LAYER



THERMAL BOUNDARY LAYER



Do you expect  $C_f$  (or  $f$ ) &  $Nu$  to be higher in the developing region or the fully developed region? Why?

**NOTES: Internal convection**

YOU CAN SEE THAT  $V = V(r)$  &  $T = T(r)$  IN THE INTERNAL FLOW CASE. LET US DEFINE, THEN



$$\dot{m} = \rho V_m A$$



$$\dot{m} c_p T_m =$$

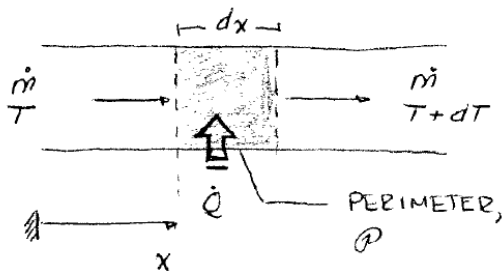


OUR **GOAL** IS TO FIND

$$\dot{Q} = hA(T_s - T_m)$$

WHAT  $T_s - T_m$  DO I USE?

TAKE A SMALL SLICE OF PIPE



Cons of Energy →

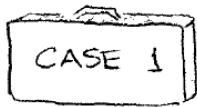
$$\frac{dE}{dt} = \dot{Q}_{in} + \dot{W}_{in} + \sum_{in} \dot{m}(h + \dots) - \sum_{out} \dot{m}(h + \dots)$$

**NOTES: Internal convection**

$\dot{q}_b$  ALSO  $\Rightarrow$   $\dot{q}_b =$  [EQN 2]

COMBINING [1] & [2]

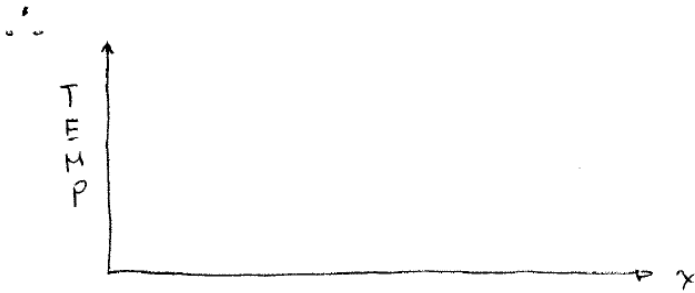
[EQN 3]



$\dot{q}_b = \text{CONST}$

[1] SAYS

[2] SAYS  
(IF  $h = \text{CONST}$ )



$\dot{q}_b =$   
 $\dot{Q} =$

$\dot{q}_b = \text{CONST}$   
BOUNDARY CONDITION

**NOTES: Internal convection**

CASE 2

$T_s = \text{CONST}$

[3] SAYS



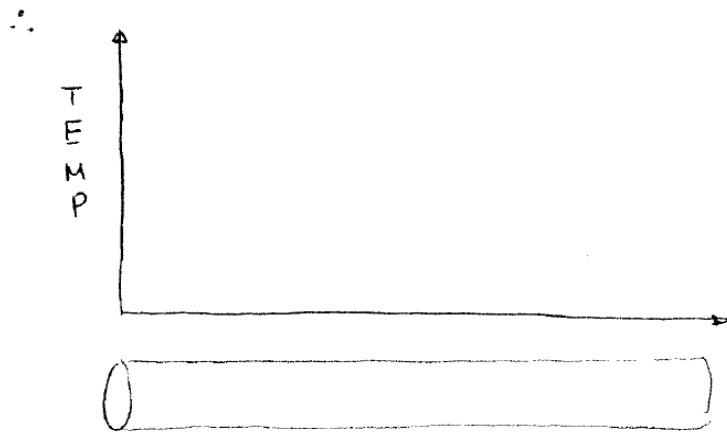
A LITTLE TRICK

$dT_m = - d(T_s - T_m)$

SO

$T_m(x) =$

[EQN 5]

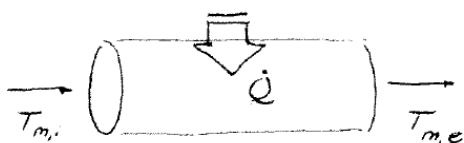


WE SEE THAT  $\dot{Q} = hA(T_s - T_m)$  IS A PROBLEM.

LET'S USE  $\dot{Q} = hA \Delta T_{\text{AVG}}$

What is  $\Delta T_{\text{AVG}}$ ?

Cons. of Energy on whole tube →



$\dot{Q} =$

[EQN 6]

**NOTES: Internal convection**

[5] FOR THE TUBE EXIT (@  $x=L$ ) GIVES

[EQN 7]

COMBINE [6] & [7] TO ELIMINATE  $\dot{m}c_p$

$$\dot{Q} = hA \left[ \frac{(T_s - T_{m,e}) - (T_s - T_{m,i})}{\ln \frac{T_s - T_{m,e}}{T_s - T_{m,i}}} \right]$$

$$\dot{Q} = hA$$

