ROSE-HULMAN INSTITUTE OF TECHNOLOGY

ME301 – Applications of Thermodynamics

Circle	section:	

 01 [4th Lui]
 02 [5th Lui]

 03 [4th Thom]
 04 [5th Thom]

 05 [4th Mech]
 04

Exam 1

Sep 29, 2017

Rules:

- Closed book/notes exam.
- Help sheet allowed. (8-1/2 x 11" sheet of paper, one side, handwritten; may not contain worked out example problems)
- Maple, Excel, and MATLAB are allowed on your laptop, but, nothing may be prepared before the exam.
- Either open property tables (hardcopies) or open Property Calculator Programs on your laptop.

Instructions:

- Show all work for complete credit.
- Work in symbols first, plugging in numbers and performing calculations last.



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Name

PROBLEM 1 [26 points]

(a) [20 pts] Indicate whether each of the following statements is true or false.

- True | False Anything that generates entropy also destroys exergy.
- **True | False** Exergy and energy have the same dimensions.
- **True** | **False** If a substance is at the same temperature as the environment (i.e., $T = T_0$) then it necessarily has no exergy.
- True | False Exergy is a conserved property.
- True | False "Isentropic efficiency" and "exergetic efficiency" are necessarily the same thing.
- True | False Based on its definition, exergy cannot be negative.
- **True | False** When heat is transferred into a system, it necessarily transfers entropy into the the system.
- **True | False** When heat is transferred into a system, it necessarily transfers exergy into the the system.
- **True | False** Exergy destruction is a consequence of the Second Law of Thermodynamics.
- True | False A turbine with 101% isentropic efficiency is thermodynamically impossible.

- (b) [3 pts] Heat transfer in the amount of 100 kJ is transferred to air in a rigid container. The reservoir is at T_R = 500 K and the environment are at T_0 = 300 K and P_0 = 100 kPa. How much exergy has been transferred to the air?
 - o 100 kJ
 - o -100 kJ
 - o 40 kJ
 - o -40 kJ
 - o 20 kJ
 - o Insufficient information to determine

- (c) [3 pts] Power in the amount of 100 kW is transferred to air in a rigid container in a steadystate process. The environment are at T_0 = 300 K and P_0 = 100 kPa. How much exergy has been transferred to the air?
 - o 100 kW
 - o -100 kW
 - o 66.7 kW
 - o -66.7 kW
 - o 33.3 kW
 - o Insufficient information to determine

PROBLEM 2 [37 points]

A mass of 0.75 kg of saturated liquid steam is **initially** contained in a piston cylinder initially at $P_1 = 200$ kPa. Heat transfer in the amount of 990.8 kJ is added to the steam in a constant pressure process **until** the quality reaches $x_2 = 0.6$. The heat is transferred from a thermal reservoir maintained at $T_R = 227^{\circ}$ C while holding the pressure of the steam constant. The environment is at $P_0 = 100$ kPa and $T_0 = 300$ K.

- (a) Sketch the process on P-v and T-s diagrams.
- (b) Calculate the work out of the steam in kJ.
- (c) Calculate the <u>useful work out of the steam</u> in kJ.
- (d) Using the Accounting of Exergy Equation, find the total exergy destroyed in the process.



Problem 3 [37 points]

A flow of \dot{m}_{air} = 10 kg/s of air is compressed adiabatically in a compressor with an isentropic (adiabatic) efficiency of η_{C} =0.8. The exiting air is then expanded *irreversibly* through a throttle valve to a lower pressure. The valve operates adiabatically and *has no moving parts*. The apparatus and pertinent property values are shown in the diagram at the right.

Treat air as an ideal gas with variable specific heats. The environment is at $P_0 = 100$ kPa and $T_0 = 27^{\circ}$ C.

- (a) Sketch the two-step process on a *<u>T-s</u> diagram*.
- (b) Calculate the compressor input power in kW.
- (c) Calculate the <u>exergetic efficiency of the compressor</u>.
- (d) *Using any reasonable method,* find the total <u>rate of exergy destruction</u> in the valve. Remember that the valve operates adiabatically and *has no moving parts*.



Length

1 ft = 12 in = 0.3048 m = 1/3 yd 1 m = 100 cm = 1000 mm = 39.37 in = 3.2808 ft 1 mile = 5280 ft = 1609.3 m

Mass

1 kg = 1000 g = 2.2046 lbm 1 lbm = 16 oz = 0.45359 kg 1 slug = 32.174 lbm

Temperature Values

 $(T/K) = (T/ \circ R) / 1.8$ $(T/K) = (T/ \circ C) + 273.15$ $(T/\circ C) = [(T/ \circ F) - 32]/1.8$ $(T/\circ R) = 1.8(T/K)$ $(T/\circ R) = (T/ \circ F) + 459.67$ $(T/ \circ F) = 1.8(T/ \circ C) + 32$

Temperature Differences

$$(\Delta T/^{\circ} R) = 1.8(\Delta T/K)$$

 $(\Delta T/^{\circ} R) = (\Delta T/^{\circ} F)$
 $(\Delta T/K) = (\Delta T/^{\circ} C)$

Volume

 $1 m^{3} = 1000 L = 10^{6} cm^{3} = 10^{6} mL = 35.315 ft^{3}$ = 264.17 gal $1 ft^{3} = 1728 in^{3} = 7.4805 gal = 0.028317 m^{3}$ $1 gal = 0.13368 ft^{3} = 0.0037854 m^{3}$

$= 4.4482 \,\mathrm{N}$

Pressure

Force

1 atm = 101.325 kPa = 1.01325 bar = 14.696 lbf/in² 1 bar = 100 kPa = 10⁵ Pa 1 Pa = 1 N/m² = 10⁻³ kPa 1 lbf/in² = 6.8947 kPa = 6894.7 N/m²; [lbf/in² = "psi"]

 $1 \text{ m}^3/\text{s} = 35.315 \text{ ft}^3/\text{s} = 264.17 \text{ gal/s}$ $1 \text{ ft}^3/\text{s} = 1.6990 \text{ m}^3/\text{min} = 7.4805 \text{ gal/s}$

= 448.83 gal/min

 $1 \text{ N} = 1 \text{ kg m/s}^2 = 0.22481 \text{ lbf}$

 $1 \text{ lbf} = 1 \text{ slug ft/s}^2 = 32.174 \text{ lbm ft/s}^2$

<u>Energy</u>

1 J = 1 N m 1 kJ = 1000 J = 737.56 ft lbf = 0.94782 Btu 1 Btu = 1.0551 kJ = 778.17 ft lbf 1 ft lbf = 1.3558 J

Energy Transfer Rate

1 kW = 1 kJ/s = 737.56 ft ·lbf/s = 1.3410 hp = 0.94782 Btu/s 1 Btu/s = 1.0551 kW = 1.4149 hp = 778.17 ft ·lbf/s 1 hp = 550 ft ·lbf/s = 0.74571 kW = 0.70679 Btu/s

Specific Energy

1 kJ/kg = 1000 m²/s² 1 Btu/lbm = 25037 ft²/s² 1 ft·lbf /lbm = 32.174 ft²/s²

Volumetric Flow Rate