
Name_____
CM**Exam 1**

Sep 20, 2012

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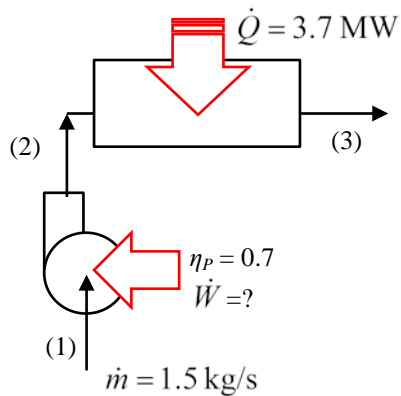
Show all work for full credit.

Open book, computer use for computational purposes, one 8 ½ x 11" handwritten equation sheet.

If you use tabular data from your text, do **not** interpolate values. Use the nearest value in the table(s).

Problem 1 (34 pts)

A pump with isentropic (adiabatic) efficiency of $\eta_p = 0.7$ compresses $\dot{m} = 1.5$ kg/s of saturated liquid water from $P_1 = 200$ kPa to $P_2 = 2.5$ MPa. The water is then passed through a constant pressure steam generator which receives heat transfer in the amount of $\dot{Q} = 3.7$ MW. The temperature and pressure of the surroundings are $T_0 = 300$ K and $P_0 = 100$ kPa. The properties at state (1) are given below.

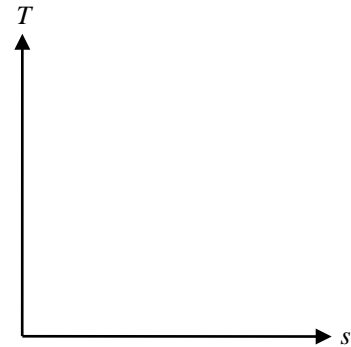


$$(1) P_1 = 200 \text{ kPa}$$

Saturated liquid
 $T_1 = 120.2^\circ\text{C}$
 $v_1 = 0.0011 \text{ m}^3/\text{kg}$
 $u_1 = 504.5 \text{ kJ/kg}$
 $h_1 = 504.7 \text{ kJ/kg}$
 $s_1 = 1.530 \text{ kJ/kg-K}$

$$(2) P_2 = 2.5 \text{ MPa}$$

$$(3) P_3 = P_2$$



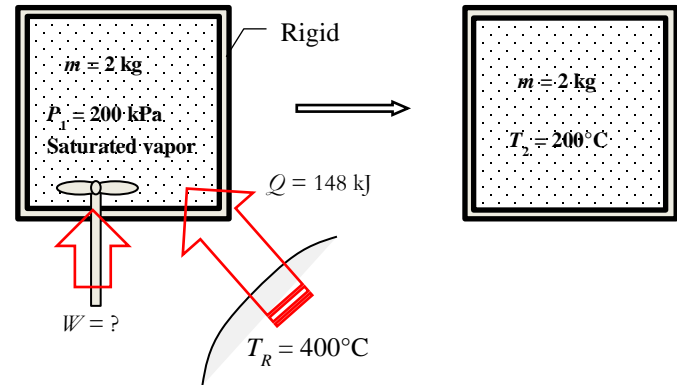
- Sketch the two step process on the T - s diagram above.
- Find the required power into the pump, \dot{W}_{in} , in kW.
- Find the specific enthalpy of the steam leaving the steam generator, h_3 , in kJ/kg and state the phase (liquid, superheated vapor, saturated mixture, etc.) of the steam leaving the steam generator.
- Find the exergetic efficiency (second law efficiency) of the pump, ϵ_p .

Problem 2 (34 pts)

A **rigid** vessel contains 2 kg of steam. Initially the steam is a **saturated vapor** at $P_1 = 200$ kPa. Heat transfer from a reservoir at $T_R = 400^\circ\text{C}$ is then added to the steam in the amount of $Q = 148$ kJ, along with an unknown amount of paddle wheel work, W , until the steam is at $T_2 = 200^\circ\text{C}$. The temperature and pressure of the surroundings are $T_0 = 300$ K and $P_0 = 100$ kPa, respectively.

Find

- the work the paddle wheel does on the steam (in kJ) and
- the change in exergy (in kJ) of the steam.
- Using the results of (a) and (b), find the amount of exergy destroyed (in kJ) in the process.



Problem 3 (32 pts)

- (a) A turbine with an adiabatic efficiency (isentropic efficiency) of $\eta_T < 1$ operates between pressures of 10 MPa and 50 kPa. The inlet is a saturated vapor.
- i. (3 pts) How does the actual exit pressure compare to the isentropic exit pressure?
 - $P_2 < P_{2s}$
 - $P_2 = P_{2s}$
 - $P_2 > P_{2s}$
 - Insufficient info to determine
 - ii. (3 pts) How does the actual exit specific entropy compare to the isentropic exit specific entropy?
 - $s_2 < s_{2s}$
 - $s_2 = s_{2s}$
 - $s_2 > s_{2s}$
 - Insufficient info to determine
 - iii. (3 pts) How does the actual exit temperature compare to the isentropic exit temperature?
 - $T_2 < T_{2s}$
 - $T_2 = T_{2s}$
 - $T_2 > T_{2s}$
 - Insufficient info to determine
- (b) (3 pts) The work out of a closed system is $W_{12} = 100$ kJ. The system's volume increases from $V_1 = 1$ m³ to $V_2 = 2$ m³ during the work addition. If the environment is at $T_0 = 300$ K and $P_0 = 100$ kPa, how much exergy has left the system because of work?
- $A_{W,\text{out}} = 0$ kJ
 - $A_{W,\text{out}} = 100$ kJ
 - $A_{W,\text{out}} = 200$ kJ
 - Insufficient info to determine
- (c) An electrical resistance heater operates at steady-state with an input electrical power of $\dot{W}_{in} = 5$ kW. The heater converts all of the electrical power into heat transfer, which is dissipated to the heated space. The temperature of the heating element is $T_{elem} = 700$ K and the heated space is at $T_{space} = 300$ K and $P_{space} = 100$ kPa.
- i. (2 pts) What is the **energy-based efficiency** of the heater?
 - $\eta_{\text{heat}} = 0\%$
 - $\eta_{\text{heat}} = 43\%$
 - $\eta_{\text{heat}} = 57\%$
 - $\eta_{\text{heat}} = 100\%$
 - Insufficient info to determine
 - ii. (4 pts) What is the **exergetic efficiency** of the heater?
 - $\epsilon_{\text{heat}} = 0\%$
 - $\epsilon_{\text{heat}} = 43\%$
 - $\epsilon_{\text{heat}} = 57\%$
 - $\epsilon_{\text{heat}} = 100\%$
 - Insufficient info to determine

(d) Circle the correct response.

- i. (3 pts) **True / False** For a turbine operating at steady-state, the total power produced and the useful power are the same.
- ii. (3 pts) **True / False** A change in exergy can be negative.
- iii. (3 pts) **True / False** Exergy destruction can be negative.

(e) (5 pts) Hot combustion gases at $T_R = 700^\circ\text{C}$ provide 5 MW of heat transfer to steam at $T_{boiler} = 600^\circ\text{C}$ in a Rankine cycle. If the boiler temperature were raised to $T_{boiler} = 650^\circ\text{C}$ instead, would you expect the cycle efficient to go up, down, or remain the same? Explain your answer.