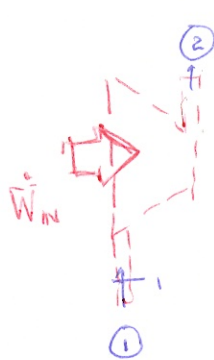
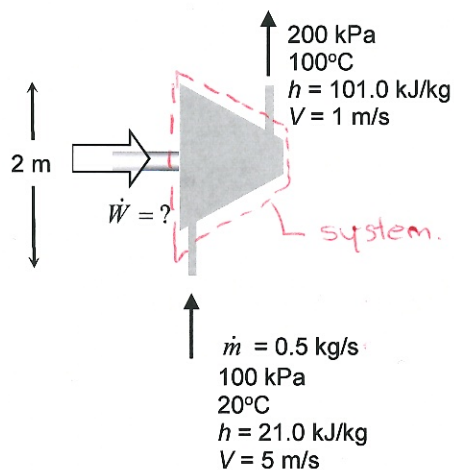


Example

0.5 kg/s of air flows steadily through a compressor. The air enters and exits the compressor at the states shown in the figure. If the compression is **adiabatic** (buzza buzza buzz) calculate the power input to the compressor.



C.O.E. → Steady state

$$\frac{d}{dt}(E_{sys}) = \dot{Q}_{IN} + \dot{W}_{IN}$$

→ 0 ADIABATIC

$$+ \sum_{IN} \dot{m} \left(h + \frac{V^2}{2} + gZ \right) - \sum_{OUT} \dot{m} \left(h + \frac{V^2}{2} + gZ \right)$$

Set to 0.

$$0 = \dot{W}_{IN} + \dot{m} \left(h_1 + \frac{V_1^2}{2} + gZ_1 \right) - \dot{m} \left(h_2 + \frac{V_2^2}{2} + gZ_2 \right)$$

$$\dot{W}_{IN} = \dot{m} \left(h_2 - h_1 + \frac{V_2^2 - V_1^2}{2} + g(Z_2 - Z_1) \right)$$

$$= 0.5 \frac{\text{kg}}{\text{s}} \left[(101.0 - 21.0) \frac{\text{KJ}}{\text{kg}} + \frac{1^2 - 5^2}{2} \frac{\text{m}^2}{\text{s}^2} \left\langle \frac{\text{J/kg}}{\text{m}^2/\text{s}^2} \right\rangle \left\langle \frac{\text{KJ}}{1000 \text{ J}} \right\rangle + 9.81 \frac{\text{m}}{\text{s}^2} \cdot [2 \text{ m}] \left\langle \frac{\text{J/kg}}{\text{m}^2/\text{s}^2} \right\rangle \left\langle \frac{\text{KJ}}{1000 \text{ J}} \right\rangle \right]$$

$$= \left(0.5 \frac{\text{kg}}{\text{s}} \right) \left(80 \frac{\text{KJ}}{\text{kg}} - 0.012 \frac{\text{KJ}}{\text{kg}} + 0.0197 \frac{\text{KJ}}{\text{kg}} \right)$$

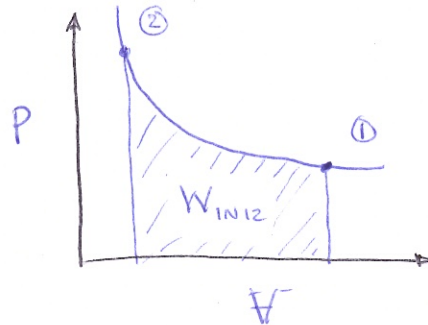
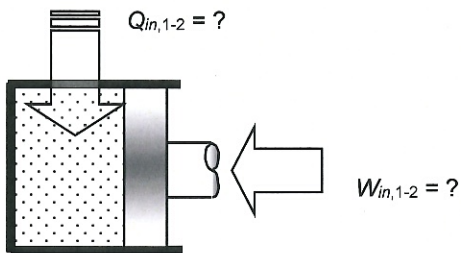
Notice these are orders of magnitude smaller.

$$= \boxed{40.0 \text{ KW}}$$

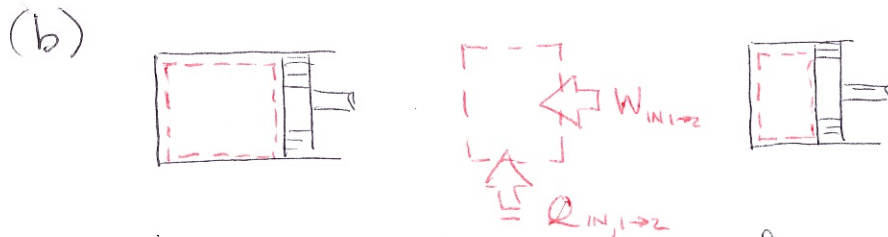
Example

0.3 kg of air is contained in a piston-cylinder assembly. Initially, the air is at 200 kPa and 20°C with a volume of $V_1 = 0.126 \text{ m}^3$. The air is then compressed in a process for which $pV^{1.2} = \text{constant}$ until the pressure is 500 kPa.

- Sketch the p - V diagram and calculate the work (in kJ) into the piston cylinder.
- If the change in *specific* internal energy during the process is 121.0 kJ/kg, calculate the heat transfer (in kJ) into the piston cylinder during the process.



$$\begin{aligned}
 (a) \quad \bar{W}_{in,1-2} &= - \int_1^2 p dV = - \int_1^2 \frac{\text{CONST}}{V^{1.2}} dV \\
 &= - \text{CONST} \int_1^2 \frac{1}{V^{1.2}} dV = - \text{CONST} \left[-V^{-0.2} \right]_1^2 = \text{CONST} \left[\frac{1}{V_2} - \frac{1}{V_1} \right] \\
 &= P_1 V_1^{1.2} \left[\frac{1}{V_2} - \frac{1}{V_1} \right] = P_1 V_1 \left[\frac{V_1}{V_2} - 1 \right] = P_1 V_1 \left[\left(\frac{P_2}{P_1} \right)^{1/1.2} - 1 \right] \\
 &= (200 \text{ kPa})(0.126 \text{ m}^3) \left[\left(\frac{500}{200} \right)^{1/1.2} - 1 \right] = \boxed{14.6 \text{ kJ}}
 \end{aligned}$$



Cons. of energy, closed system, finite time

$$E_2 - E_1 = Q_{in,1-2} + W_{in,1-2}$$

Only internal energy is important:

$$U_2 - U_1 = Q_{in,1-2} + W_{in,1-2}$$

$$m(u_2 - u_1) = Q_{in,1-2} + W_{in,1-2}$$

$$Q_{in,1-2} = m(u_2 - u_1) - W_{in,1-2} = (0.3 \text{ kg}) \left(121.0 \frac{\text{kJ}}{\text{kg}} \right) - 14.6 \text{ kJ} = \boxed{21.8 \text{ kJ}}$$