ECE-320: Linear Control Systems Homework 6

Due: Monday April 25 *at the beginning of class*

1) Consider a system with closed loop transfer function $G_o(s) = \frac{a\kappa_p}{s}$ *p* $G_o(s) = \frac{\alpha k}{s}$ $s + \alpha + k$ α $=\frac{am_p}{s+\alpha+k_p}$. The *nominal* values for the

parameters are $k_p = 1$ and $\alpha = 2$.

- a) Determine an expression for the sensitivity of the closed loop system to variations in k_p . Your final answer should be written as numbers and the complex variable *s*.
- b) Determine an expression for the sensitivity of the closed loop system to variations in α . Your final answer should be written as numbers and the complex variable *s*.
- c) Determine expressions for the *magnitude* of the sensitivity functions in terms of frequency,
- d) As $\omega \rightarrow \infty$ the system is more sensitive to which of the two parameters?

2) Consider the plant

$$
G_p(s) = \frac{\alpha_0}{s + \alpha_1} = \frac{3}{s + 0.5}
$$

where 3 is the nominal value of α_0 and 0.5 is the nominal value of α_1 . In this problem we will investigate the sensitivity of closed loop systems with various types of controllers to these two parameters. We will assume we want the settling time of our system to be 0.5 seconds and the steady state error for a unit step input to be less than 0.1.

a) (*ITAE Model Matching*) Since this is a first order system, we will use the first order ITAE model,

$$
G_o(s) = \frac{\omega_o}{s + \omega_o}
$$

i) For what value of ω _o will we meet the settling time requirements and the steady state error requirements?

ii) Determine the corresponding controller $G_c(s)$.

iii) Show that the closed loop transfer function (using the parameterized form of $G_p(s)$ and the controller from part ii) is

$$
G_o(s) = \frac{\frac{8}{3}\alpha_0(s+0.5)}{s(s+\alpha_1) + \frac{8}{3}\alpha_0(s+0.5)}
$$

iv) Show that the sensitivity of $G_o(s)$ to variations in α_0 is given by $S_{\alpha_0}^{G_0} = \frac{S}{s+8}$ $S_{\alpha}^{G_0} = \frac{S}{\cdots}$ $\frac{G_0}{\alpha_0} = -\frac{1}{S}$ $^{+}$

v) Show that the sensitivity of $G_o(s)$ to variations in α_1 is given by $S_{\alpha_1}^{G_o} = \frac{1}{s^2}$ 0.5 $S_{\alpha_1}^{G_o} = \frac{-0.5s}{s^2 + 8.5s + 4}$ $=\frac{-0.5s}{s^2+8.5s+4}$ b) (*Proportional Control*) Consider a proportional controller, with $k_p = 2.5$.

i) Show that the closed loop transfer function is $G_0(s) = \frac{2.5a_0}{s}$ $a_1 + 2.5a_0$ $(s) = \frac{2.5}{s}$ $G_o(s) = \frac{2.5\alpha_0}{s + \alpha_1 + 2.5}$ $=\frac{2.5a_0}{s+\alpha_1+2.5\alpha_0}$ ii) Show that the sensitivity of $G_o(s)$ to variations in α_0 is given by $S_{\alpha_0}^{G_o}$ $\bf{0}$ 0.5 8 $S_{\alpha}^{G_0} = \frac{S}{\alpha}$ α_0 *S* $=\frac{s+}{}$ $^{+}$ iii) Show that the sensitivity of $G_o(s)$ to variations in α_1 is given by $S^G_{\alpha_1}$ 0.5 8 $S^{G_o}_{\alpha}$ a_1 *s* $=$ $\frac{-}{-}$ $^{+}$

c) (*Proportional+Integral Control*) Consider a PI controller with $k_p = 4$ and $k_i = 40$.

i) Show that the closed loop transfer function is $G_0(s) = \frac{4a_0}{s}$ t_1) + 4 α_0 $s = \frac{4\alpha_0 (s+10)}{s(s+10)}$ $G_o(s) = \frac{4\alpha_0(s+10)}{s(s+\alpha_1)+4\alpha_0(s+10)}$ $=\frac{4\alpha_0(s+10)}{s(s+\alpha_1)+4\alpha_0(s+10)}$

ii) Show that the sensitivity of $G_o(s)$ to variations in α_0 is given by $S_{\alpha_0}^{G_0} = \frac{1}{s^2}$ $(s+0.5)$ $\frac{1}{12.5s + 120}$ $S_{\alpha_0}^{G_0} = \frac{s(s)}{s^2}$ $\frac{a_0}{a_0} = \frac{1}{s^2 + 12.5s}$ $=\frac{s(s+0.5)}{s^2+12.5s+12}$

iii) Show that the sensitivity of $G_o(s)$ to variations in α_1 is given by $S_{\alpha_1}^{G_o} = \frac{1}{s^2}$ 0.5 $S_{\alpha_1}^{G_o} = \frac{-0.5s}{s^2 + 12.5s + 120}$ $=\frac{-0.5s}{s^2+12.5s+12}$

d) Using Matlab, simulate the unit step response of each type of controller. Plot all responses on one graph. Use different line types and a legend. Turn in your plot and code. *Do not make separate graphs for each system!*

e) Using Matlab and subplot, plot the sensitivity to α_0 for each type of controller on *one graph* at the top of the page, and the sensitivity to α_1 on one graph on the bottom of the page. Be sure to use different line types and a legend. Turn in your plot and code. Only plot up to about 8 Hz (50 rad/sec) using a semilog scale with the sensitivity in dB (see next page). *Do not make separate graphs for each system!*

In particular, these results should show you that the model matching method, which essentially tries and cancel the plant, are generally more sensitive to getting the plant parameters correct than the PI controller for low frequencies. However, for higher frequencies the methods are all about the same.

Hint: If $T(s) = \frac{1}{s^2}$ $(s) = \frac{2}{2a+2}$ $\frac{2s+10}{ }$ $T(s) = \frac{2s}{s^2}$ $=\frac{2s}{s^2+2s+10}$, plot the magnitude of the frequency response using:

T = tf([2 0],[1 2 10]); w = logspace(-1,1.7,1000); $[M,P]=bode(T,w);$ $Mdb = 20 * log10(M(:));$ *semilogx(w,Mdb); grid;*

xlabel('Frequency (rad/sec)'); ylabel('dB');

3) Assume $x(t) = 3 + 2\cos(2t - 3)$ is the input to an LTI system with transfer function

$$
H(j\omega) = \begin{cases} 2e^{-j\omega} & | \omega | < 3 \\ 3e^{-j2\omega} & | \omega | \ge 3 \end{cases}
$$

The **steady state output** will be

a) $y(t) = 6 + 4\cos(2t - 5)$ b) $y(t) = 4\cos(2t - 5)$ c) $y(t) = [3 + 2\cos(2t - 3)][2e^{-j\omega}]$

d) $y(t) = 6 + 4\cos(2t - 3)e^{-j2}$ e) $y(t) = 3 + 4\cos(2t - 5)$ f) none of these

4) Assume $x(t) = 2 + \sin(5t) + 3\cos(8t + 30^\circ)$ is the input to an LTI system with transfer function shown below

The **steady state output** of this system will be

a)
$$
y(t) = 20 + 5\sin(5t + 90^\circ) + 6\cos(8t + 90^\circ)
$$
 b) $y(t) = 2 + 5\sin(5t + 90^\circ) + 6\cos(8t + 90^\circ)$

c)
$$
y(t) = 20 + 5\sin(5t + 90^\circ) + 6\cos(8t + 120^\circ)
$$
 d) $y(t) = 10 + 5\sin(5t + 90^\circ) + 6\cos(8t + 120^\circ)$

e) none of these

Problems 5 and 6 refer to a system whose frequency response is represented by the Bode plot below

5) If the input to the system is $x(t) = 5\cos(10t + 30^\circ)$, then the steady state output is best estimated as

a) $y_{ss}(t) = 0$ b) $y_{ss}(t) = 5\cos(10t + 30^\circ)$

c) $y_{ss}(t) = 5\cos(10t - 20^\circ)$
d) $y_{ss}(t) = 5\cos(10t - 50^\circ)$

6) If the input to the system is $x(t) = 50\sin(100t)$, then the steady state output is best estimated as

- a) $y_{ss}(t) = -2000\sin(100t 100^\circ)$ b) $y_{ss}(t) = 0.5\sin(100t 100^\circ)$
- c) $y_{ss}(t) = 2000\sin(100t 100^{\circ})$ d) $y_{ss}(t) = 5\sin(100t 100^{\circ})$

7) For the straight line approximation to the magnitude portion of a Bode plot shown below, the best estimate of the corresponding transfer function is

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9) The following three figures display the magnitude of six transfer functions. All of the poles and zeros of these transfer functions are in the left half plane (these are minimum phase transfer functions). All of the magnitudes, poles, and zeros are either zero or simple powers of $10 (10^{-1}, 1, 10^{1}, 10^{2}$ etc). Estimate the transfer functions.

Figure 1: Problem 9, Systems a and b

Figure 2: Problem 9, Systems c and d

Figure 3: Problem 9, Systems e and f

Answers:

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$$
H(s) = \frac{100}{s^2}, \quad H(s) = \frac{0.1\left(\frac{1}{100}s + 1\right)^2}{s^2}, \quad H(s) = 10\left(\frac{1}{100}s + 1\right)
$$
\n
$$
H(s) = \frac{0.1\left(\frac{1}{10}s + 1\right)^2}{\left(\frac{1}{1000}s + 1\right)^2}, \quad H(s) = \frac{0.1\left(\frac{1}{10}s + 1\right)^2}{\left(\frac{1}{1000}s + 1\right)^2\left(\frac{1}{1000}s + 1\right)^2}, \quad H(s) = \frac{1}{\left(\frac{1}{10}s + 1\right)\left(\frac{1}{100}s + 1\right)}
$$

10) (We will be using this in lab 5!, You mostly just run code for this problem) Assume we have reason to believe that the plant we want to design a controller for has the form

$$
G_p(s) = \frac{Ks^2}{s^2 + 2\zeta\omega_n s + \omega_n^2}
$$

and we need to estimate the paramters K , ζ , and ω_n .

Since we get transfer functions from LTI systems, one way to estimate these parameters is to experimentally construct a Bode plot of the system by using inputs of known frequencies and amplitudes, and then measuring the amplitude and time delay (phase shift) of the output. Once we have the data for a Bode plot, we try and optimize the transfer function parameters to provide the best fit to the data. If this brings back nightmares of the last two labs in ECE205, then that class was not a complete waste of your time.

The Matlab program **process_data_pendulum.m** (from the class webpage) has some data collected for you for an experiment I did with the pendulum systems. Yout will be filling in the data for this program when you have your own system in the next lab. The first column is the input the frequency (in Hz), the second column is the input amplitude (0.087, in rad), the final column is the average output amplitude (in rad). Note that here we will only be estimating the magnitude portion of the Bode plot, so we are not concerned with the time delay (or phase change) between the intput and output.

In the Matlab command window, type

data = process_data_pendulum

This will write to the array **data**, and write the frequency and gain at that frequency to the screen. Remember the frequency at which the maximum gain occurs.

To estimate the parameters we need we will use the Matlab program **model_pendulum.m** (from the class website) which utilizes Matlab's **fminsearch** routine. The input arguments to

model_pendulum.m are the array data, the initial guess of the parameter *K* (assume it is 1.0), the initial guess of the parameter ζ (assume it is 0.05), and the initial guess of the parameter ω_n (assume it is the frequency of the maximum gain, converted to radians/sec.) The program spits a bunch of incredibly useful information to the screen, and eventually produces a plot of the transfer function and the measured data. The title indicates the estimated values of the parameters. If the transfer function does not seem to match the measured data very well you may want to try different initial estimates of the parameters.

Turn in your final plot once you think you have a good fit.