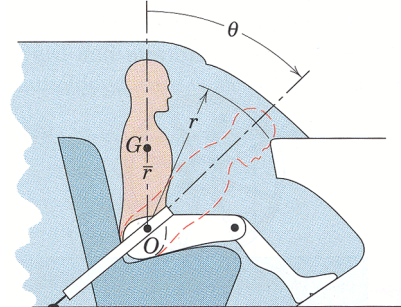


Review Problems - Final Exam

2. In a study of head injury against the dashboard of a car during sudden or crash stops where lap belts without shoulder straps are used, the segmented human model shown in the figure is analyzed. The hip joint O is assumed to remain fixed relative to the car and the torso above the hip is treated as a rigid body of mass m pinned at O. The center of mass of the torso is at G with the initial position of OG taken as vertical. The radius of gyration about O is k_O . If the car is brought to a stop with a constant deceleration a , determine the velocity v relative to the car with which the model's head strikes the dashboard. Substitute the values:



$$m = 50 \text{ kg} \quad \bar{r} = 450 \text{ mm} \quad r = 800 \text{ mm} \quad k_O = 550 \text{ mm} \quad q = 45^\circ \quad a = 10g$$

(taken from Dynamics by Meriam and Kraige, Fourth Edition)

Hint: $I_O = I_G + m\bar{r}^2$

ans: $v = 11.73 \text{ m/s}$

Strategy: Since the car is moving at a non-constant velocity, we can't use COE to obtain a solution. We'll therefore apply the rate form equations in the displaced position (right before the head hits the dashboard) and work backward.

unk	eqs
v_H	1
W	2
a_{Gx}	3
a_{Gy}	4
α_{GO}	5
ω_{GO}	6
a_{Ox}	7
a_{Oy}	8

Kinetics:

COAM(RF) at O

$$-W\bar{r} \sin q = -I_G \mathbf{a}_{GO} + m a_{Gy} \bar{r} \sin q - m a_{Gx} \bar{r} \cos q \tag{1}$$

Kinematics:

$$\begin{aligned} \bar{\mathbf{a}}_G &= \bar{\mathbf{a}}_O + \bar{\mathbf{a}}_{G/O} \\ &= \bar{\mathbf{a}}_O + \bar{\boldsymbol{\omega}}_{GO} \times \bar{\mathbf{r}}_{G/O} - \mathbf{w}_{GO}^2 \bar{\mathbf{r}}_{G/O} \end{aligned}$$

$$\begin{aligned} \hat{i}: \quad a_{Gx} &= a_{Ox} + \mathbf{a}_{GO} r_{G/O_y} - \mathbf{w}_{GO}^2 r_{G/O_x} \\ \hat{j}: \quad a_{Gy} &= a_{Oy} - \mathbf{a}_{GO} r_{G/O_x} - \mathbf{w}_{GO}^2 r_{G/O_y} \end{aligned} \tag{2,3}$$

Note the signs in (2,3) preceding the angular acceleration terms. What has happened?

Some early kinematics reminds of the relationship between angular acceleration and angular velocity

$$\mathbf{a} = \mathbf{w} \frac{d\mathbf{w}}{dq} \quad (4)$$

and once we know the angular velocity

$$v_H = \mathbf{w}_{GO} r \quad (5)$$

Constraints and Geometry:

$$\begin{aligned} \bar{a}_O &= -10g\hat{i} + 0\hat{j} \Rightarrow a_{O_x} = -10g, a_{O_y} = 0 \\ \bar{r}_{G/O} &= \bar{r} \sin q \hat{i} + \bar{r} \cos q \hat{j} \Rightarrow r_{G/O_x} = \bar{r} \sin q, r_{G/O_y} = \bar{r} \cos q \end{aligned} \quad (6,7)$$

Other:

$$W = mg \quad (8)$$

Solving:

Substituting (2,3,6,7,8) into (1) along with the position vector components and rearranging gives:

$$\mathbf{a}_{GO} = \frac{mg\bar{r}}{I_G + m\bar{r}^2} (\sin q + 10\cos q)$$

What happened to the angular velocity?

Substituting the hint and relating the mass moment of inertia to the radius of gyration

$$\mathbf{a}_{GO} = \frac{g\bar{r}}{k_o^2} (\sin q + 10\cos q)$$

Substituting into (4) and integrating

$$\int_0^{\mathbf{w}_{GO}} \mathbf{w} d\mathbf{w} = \int_0^q \mathbf{a}_{GO} dq$$

gives

$$\mathbf{w}_{GO} = 14.66 \text{ rad/s}$$

which, when substituted into (5)

$$v_H = 11.73 \text{ m/s} \quad (26 \text{ mph})$$